

Chapter 3

Synthesis of Mechanisms (Design)

Objective:

To design a mechanism to achieve Kinematic (goals)

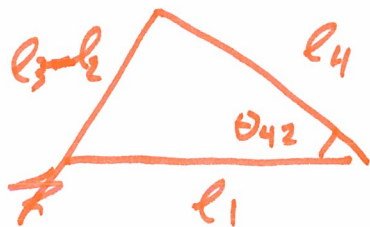
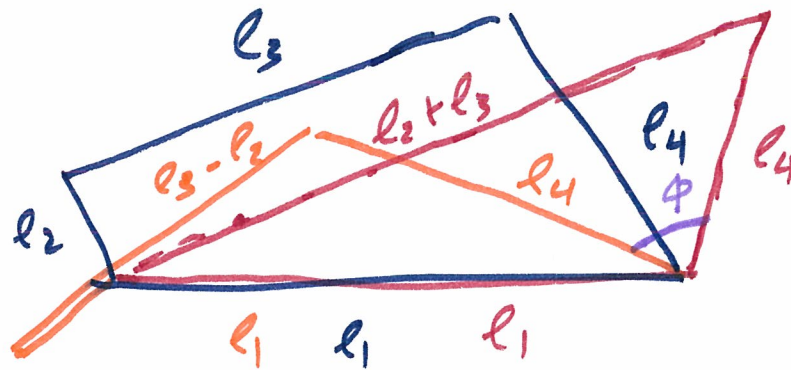
Synthesis of 4-bar mechanisms

Problem:

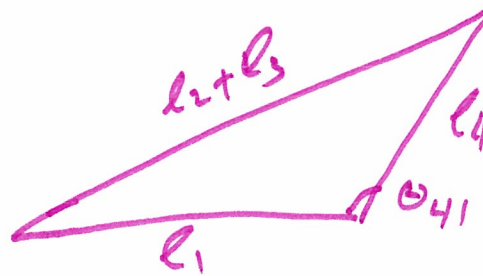
Design a 4-bar mechanism with a given oscillation angle (ϕ)

oscillation $\rightarrow$ crank-rocker mechanism
typically input is a motor rotating continuously

CR $\rightarrow$ satisfies Grashof criteria
shortest is the crank



ELP



ERP

$$\phi = \theta_{41} - \theta_{42}$$

$$\text{ELP} \quad (l_3 - l_2)^2 = l_1^2 + l_4^2 - 2 l_1 l_4 \cos \theta_{42}$$

$$\text{ERP} \quad (l_2 + l_3)^2 = l_1^2 + l_4^2 - 2 l_1 l_4 \cos \theta_{41}$$

3 equations

6 unknowns $l_1, l_2, l_3, l_4, \theta_{41}, \theta_{42}$

Solution

Provide θ_{42} l_1 l_4

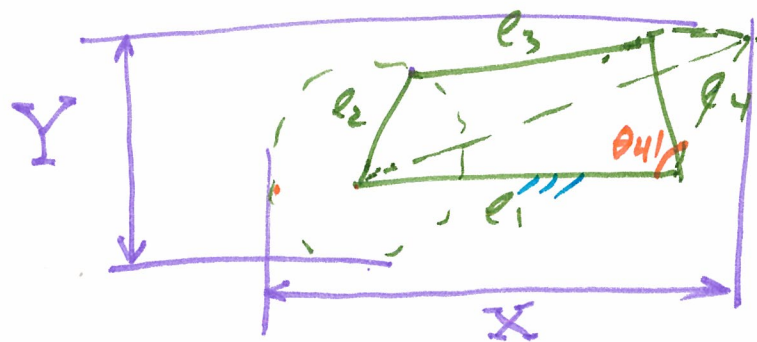
$$\theta_{41} = \phi + \theta_{42}$$

$$\begin{cases} l_3 - l_2 = [l_1^2 + l_4^2 - 2l_1 l_4 \cos \theta_{42}]^{1/2} \\ l_2 + l_3 = [l_1^2 + l_4^2 - 2l_1 l_4 \cos (\phi + \theta_{42})]^{1/2} \end{cases}$$

solve l_2 & l_3

Additional Constraint

The mechanism should fit X by Y space



space
is function
of motion
of l_2 &
E R I P

$$X = l_2 + l_1 + l_4 \cos (180 - \theta_{41})$$

$$Y = l_2 + l_4$$

If the mechanism is bigger than
the space, scale ~~it~~ ^{all links} down to fit

Example:

Given an oscillation angle of 45° ,
design a 4-bar mechanism that
produces this oscillation and can
fit a $2'' \times 1''$ rectangle

Given $\phi = 45^\circ$

Let $\theta_{42} = 45^\circ$ $l_1 = 1''$ $l_4 = \frac{3}{4}''$

$$\theta_{41} = \phi + \theta_{42} = 90^\circ$$

$$l_3 - l_2 = \left((1)^2 + (.75)^2 - 2(1)(.75)\cos(115^\circ) \right)^{1/2}$$

$$l_3 + l_2 = \left((1)^2 + (.75)^2 - 2(1)(.75)\cos(90^\circ) \right)^{1/2}$$

$$l_3 - l_2 = 1.71$$

$$l_3 + l_2 = 1.25$$

$$l_3 = \frac{1.71 + 1.25}{2} = \frac{2.96}{2} = 1.48''$$

$$0.99 + l_2 = 1.25 \rightarrow l_2 = 0.26''$$

check: Grashof

$$l_1 + l_2 < l_3 + l_4 \rightarrow \text{shortest is the crank}$$

$\downarrow$
C R

Check space need for unobstructed motion of the mechanism

$$X = l_2 + l_1 + l_4 \cos(\theta_2 180 - \theta_{41})$$

$$Y = l_2 + l_4$$

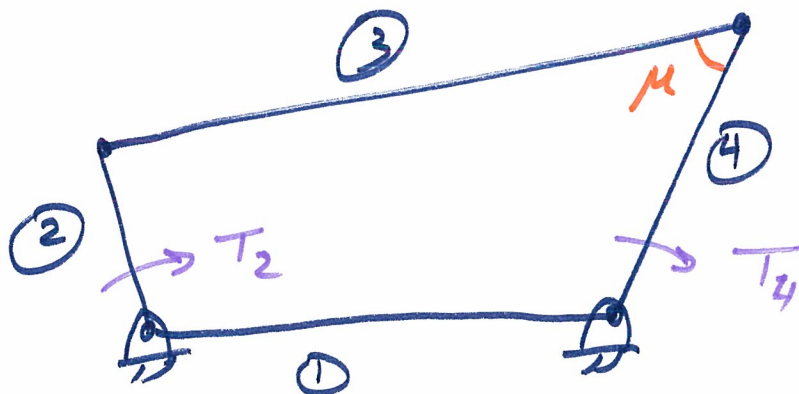
$$X = 0.26 + 1 + 0.75 \cos(180 - 90) \\ = 1.26''$$

$$Y = 0.26 + 0.75 = 1.01''$$

scale down by $\frac{1}{1.01}$ ratio

[All links]

Transmission Angle of 4-bar Mechanism

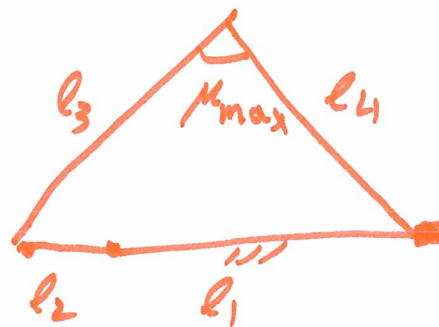
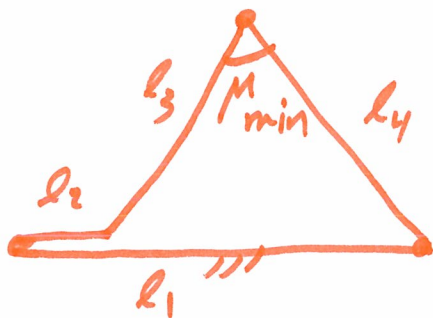


$\frac{T_4}{T_2}$ is function of μ
 $\downarrow$
 $\sin(\mu)$

- Typically, we would like to design machines with smooth output torque
- We would like μ to have reduce change range over the cycle of motion
- We would also like to ~~be~~ have μ around 90°

The general recommendation is to have $45^\circ \leq \mu \leq 135^\circ$

- Our task is to identify $\mu_{\min}$ & $\mu_{\max}$



$$(l_1 - l_2)^2 = l_3^2 + l_4^2 - 2 l_3 l_4 \cos(\mu_{\min})$$

$$(l_1 + l_2)^2 = l_3^2 + l_4^2 - 2 l_3 l_4 \cos(\mu_{\max})$$

* Objective is to compare $\mu_{\min}$ & $\mu_{\max}$ to the 45° & 135°

* Redesign if either is outside this range

Example: check μ range for the previous example

$$l_1 = 1'' \quad l_2 = 0.26'' \quad l_3 = 0.99'' \quad l_4 = 0.7''$$

$$\mu_{\min} = \cos^{-1} \left(\frac{l_3^2 + l_4^2 - (l_1 - l_2)^2}{2 l_3 l_4} \right)$$

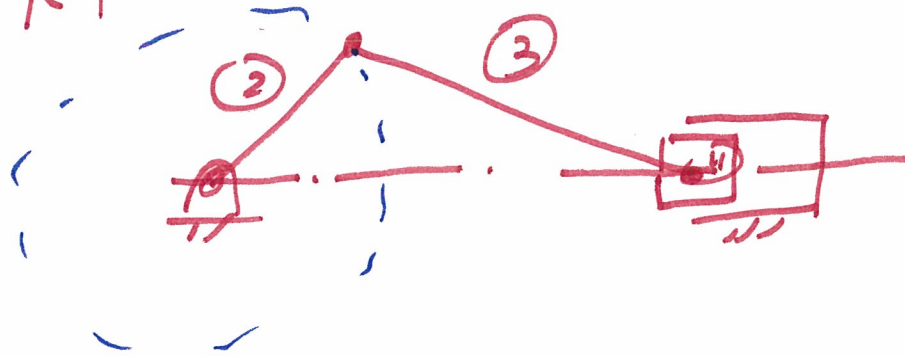
$$\mu_{\max} = \cos^{-1} \left(\frac{l_3^2 + l_4^2 - (l_1 + l_2)^2}{2 l_3 l_4} \right)$$

$$\mu_{\min} = 47.7^\circ$$

$$\mu_{\max} = 93.5^\circ \quad \left. \begin{array}{l} \mu_{\min} = 47.7^\circ \\ \mu_{\max} = 93.5^\circ \end{array} \right\} \text{OK}$$

Crank-~~slider~~ Mechanism

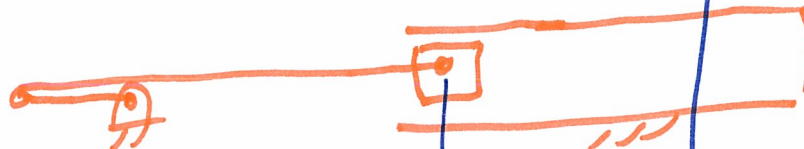
RRRP



difference
in crank
~~position~~ angles
at extreme
positions = 180°



ERP



ELP

stroke

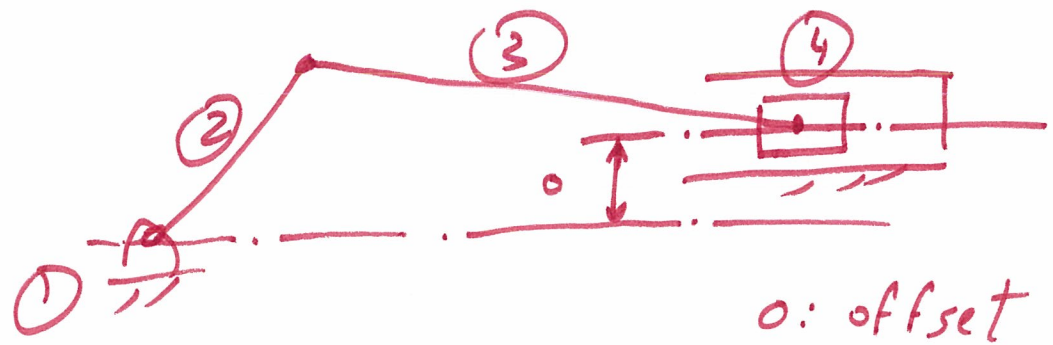
$$\text{stroke} = 2l_2$$

$$l_3 > l_2$$

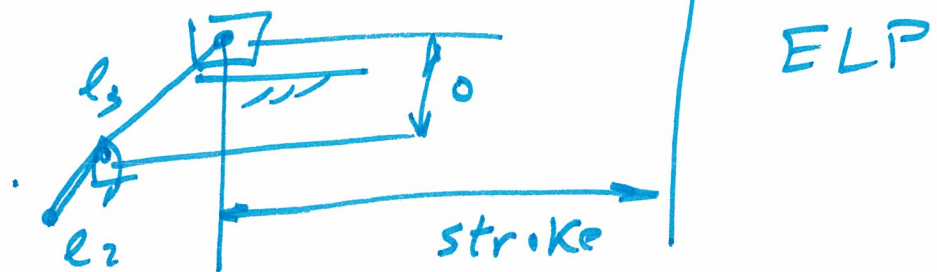
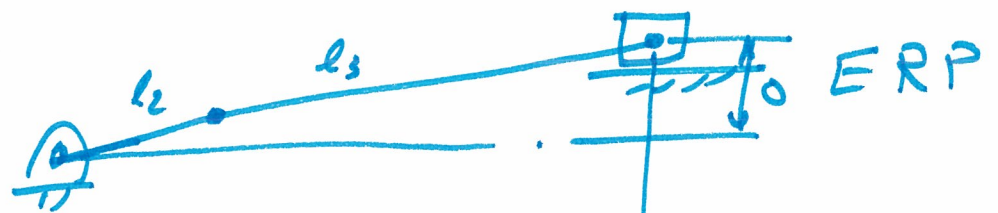


$$l_3 = l_2 \rightarrow \text{jamming}$$

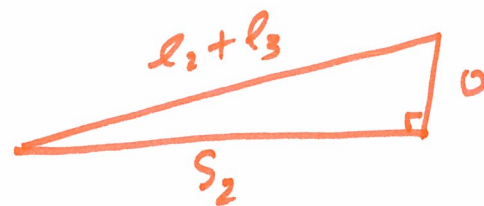
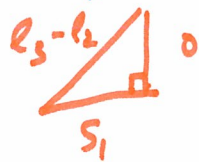
Offset Crank-Slider Mechanism



o: offset



stroke equation



$$\text{stroke} = S_2 - S_1$$

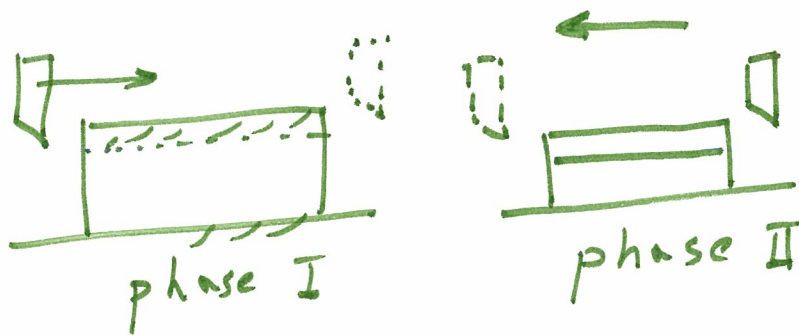
$$= \sqrt{(l_2 + l_3)^2 - o^2} - \sqrt{(l_3 - l_2)^2 - o^2}$$

Design of Quick-Return Mechanisms

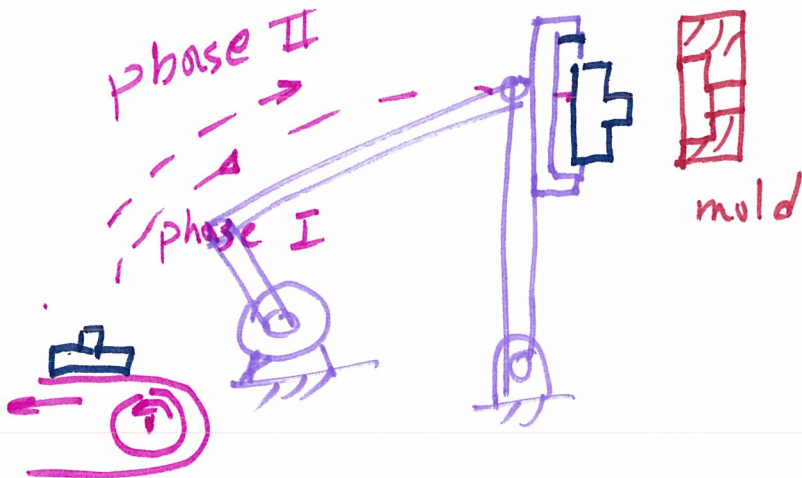
In many applications, the objective is to do a task then return to the original configuration

Examples

* Milling



* Moving product on assembly line



In all these examples, there are two phases:

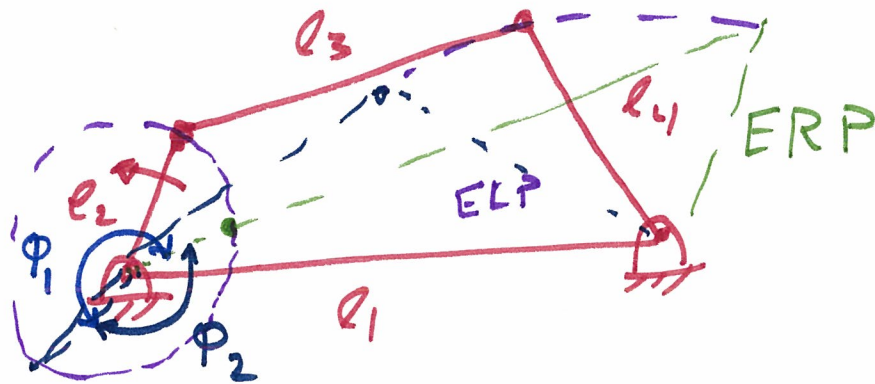
1. Power stroke (task is performed)
2. Return stroke (go back to original configurations)

* To increase productivity,
return ~~to~~ stroke duration
 $<$
power stroke duration

If input is running at a uniform velocity, can we ~~re~~ synthesize a mechanism to achieve this objective?

4-Bar Mechanism as a Quick-Return Mechanism (QRTM)

→ discussion is limited to CR



Introduce time ratio (TR)

ERP → ELP: ϕ_1

ELP → ERP: ϕ_2

$$TR = \frac{\phi_1}{\phi_2}$$

Special case: $\phi_1 = \phi_2 \rightarrow TR = 1$

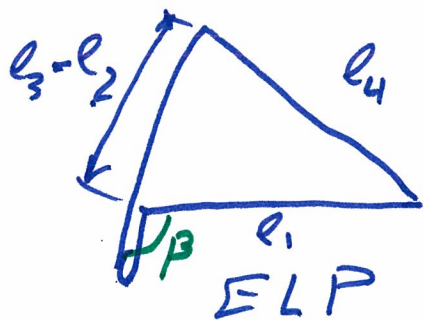
$$\phi_1 + \phi_2 = 360^\circ$$

$$TR = \frac{360 - \phi_2}{\phi_2}$$

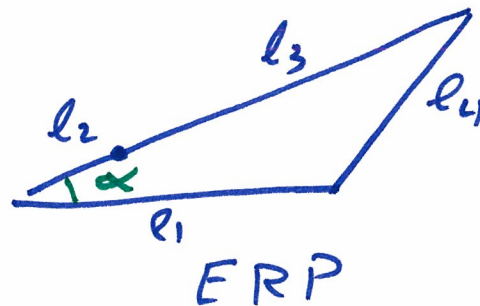
NOT a QRTM

Design Problem:

~~Given~~ ^{Design} a 4-bar mechanism with a specific time ratio



$$\phi_2 = \alpha + \beta$$



$$\text{ERP} \rightarrow l_4^2 = l_1^2 + (l_2 + l_3)^2 - 2l_1(l_2 + l_3)\cos\alpha$$

$$\text{ELP} \rightarrow l_4^2 = l_1^2 + (l_3 - l_2)^2 - 2l_1(l_3 - l_2)\cos(180 - \beta)$$

2 equations

5 unknowns $l_1, l_2, l_3, l_4, \alpha$ } provide 3 variables

if TR is known ϕ_2 ✓

→ α and β are related

Example:

Design a 4-bar mechanism with
 $TR = 2$

$$TR = \frac{360 - \phi_2}{\phi_2}$$

$$2 = \frac{360 - \phi_2}{\phi_2} \rightarrow 3\phi_2 = 360$$
$$\phi_2 = 120^\circ$$

$$\phi_2 = \alpha + \beta$$

$$\text{let } \beta = 90^\circ$$

$$\alpha = 30^\circ$$

$$l_4^2 = l_1^2 + (l_2 + l_3)^2 - 2l_1(l_2 + l_3)\cos 30$$

$$l_4^2 = l_1^2 + (l_3 - l_2)^2 - 2l_1(l_3 - l_2)\cos(180 - 90)$$

provide l_2 & $l_3 \rightarrow$ solve for: l_4 l_1

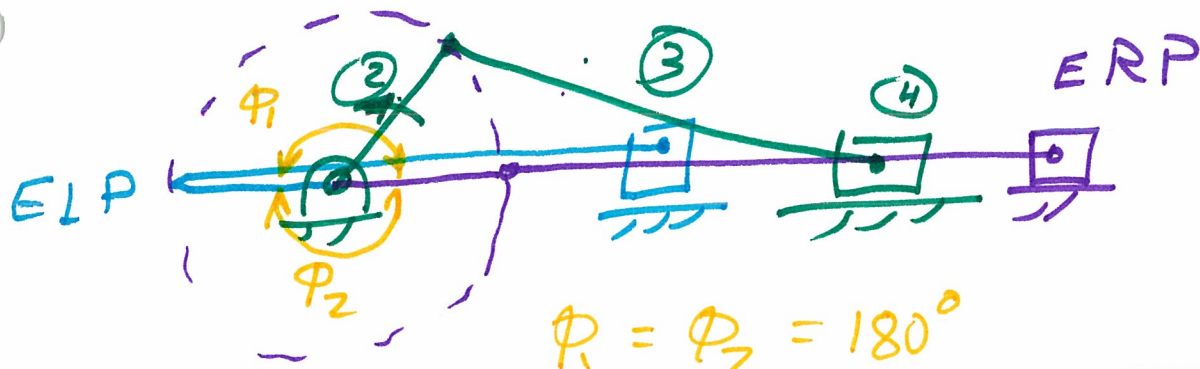
Special case if $TR=1$

$$TR = \frac{360 - \phi_2}{\phi_2} = 1$$

$$2\phi_2 = \cancel{+360} 360$$

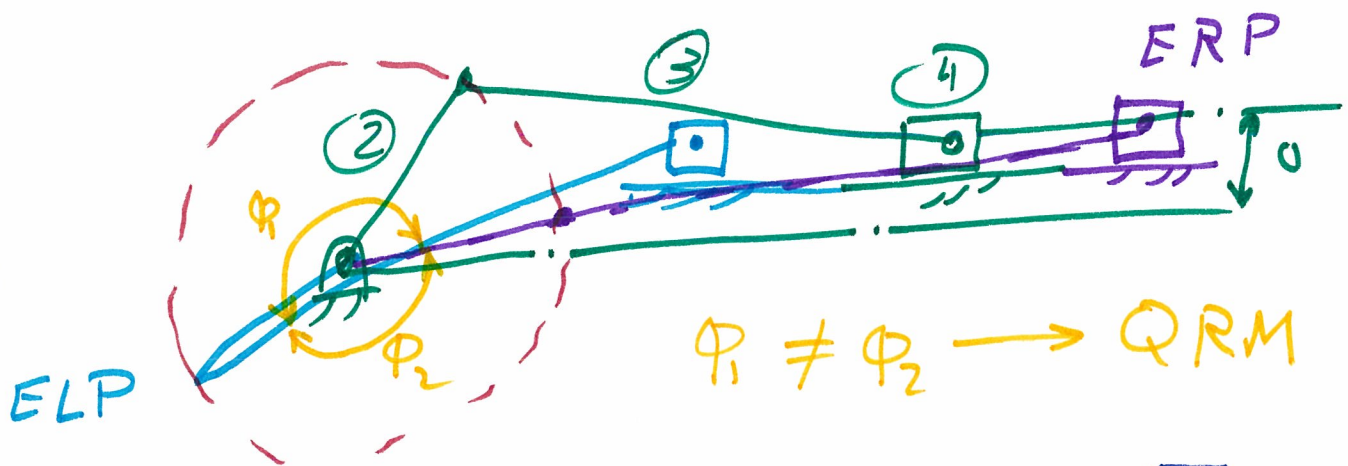
$$\phi_2 = 180^\circ$$

Crank-Slider as a QRM

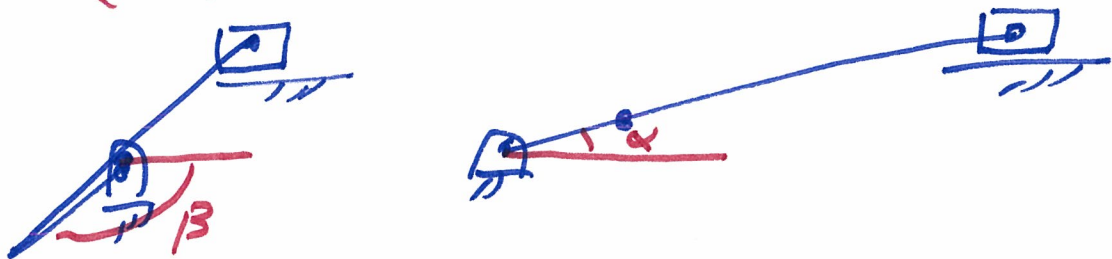


$$\Phi_1 = \Phi_2 = 180^\circ$$

$TR = 1 \rightarrow$ WOT a QRM



$\Phi_1 \neq \Phi_2 \rightarrow$ QRM

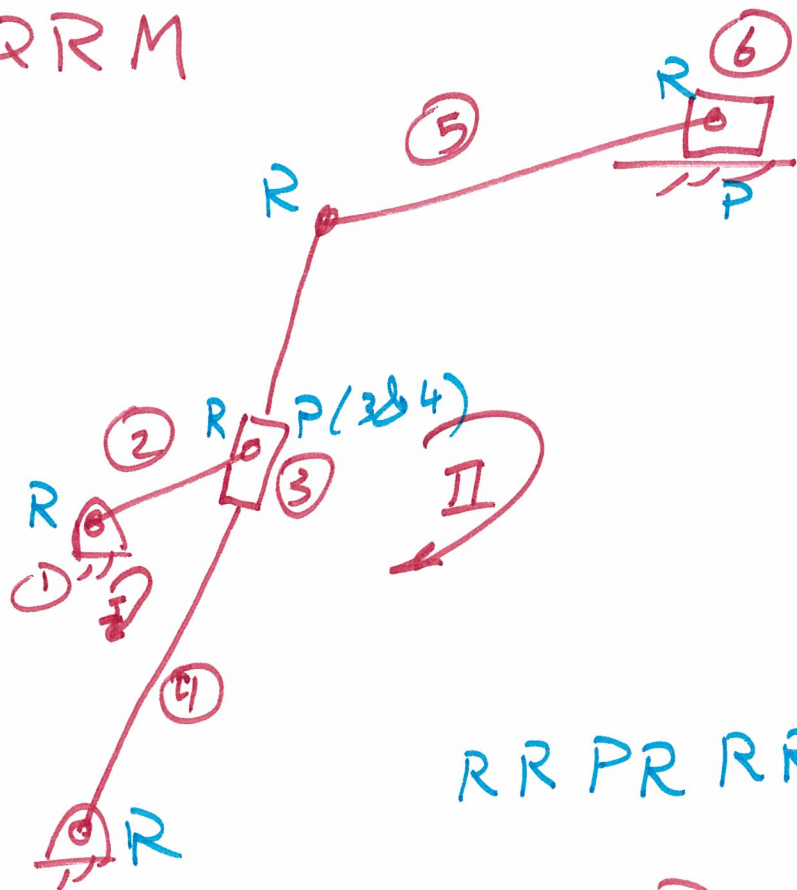
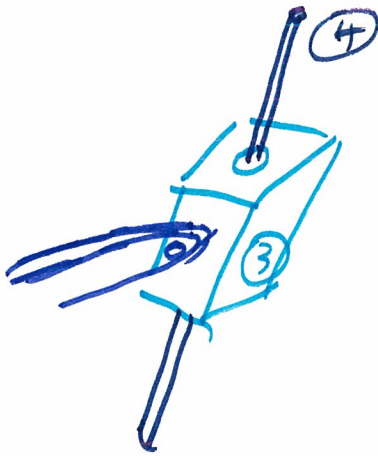


HW Design a QRM using offset crank-slider with the following specifications:

$$TR = 2$$

$$\text{stroke} = 4''$$

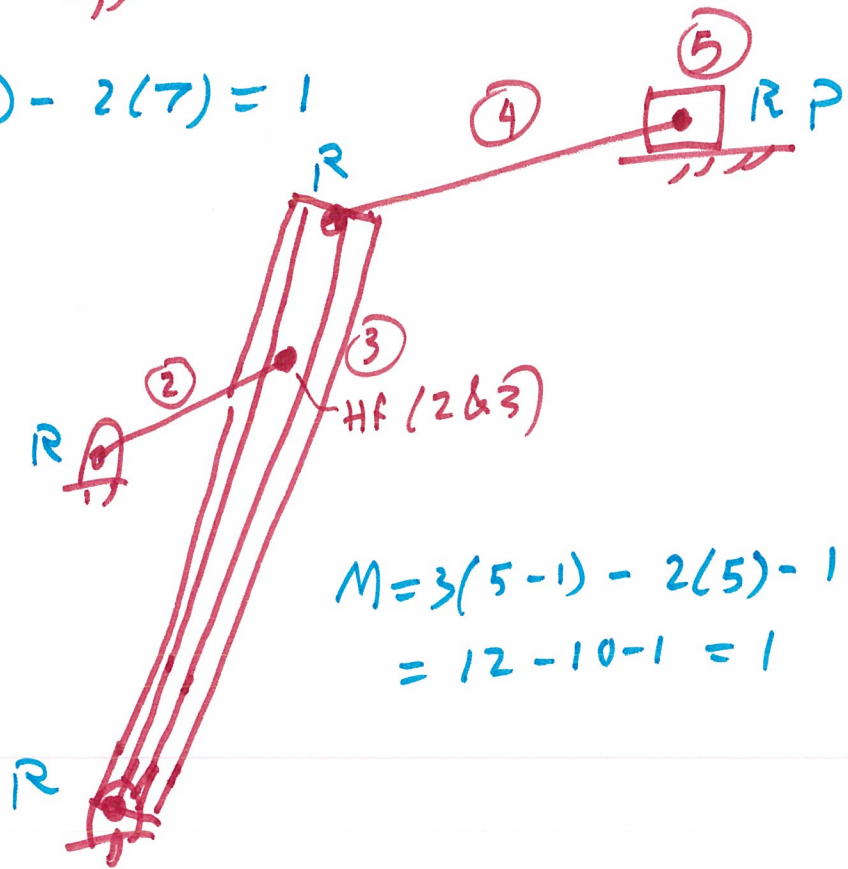
Shaper QRM



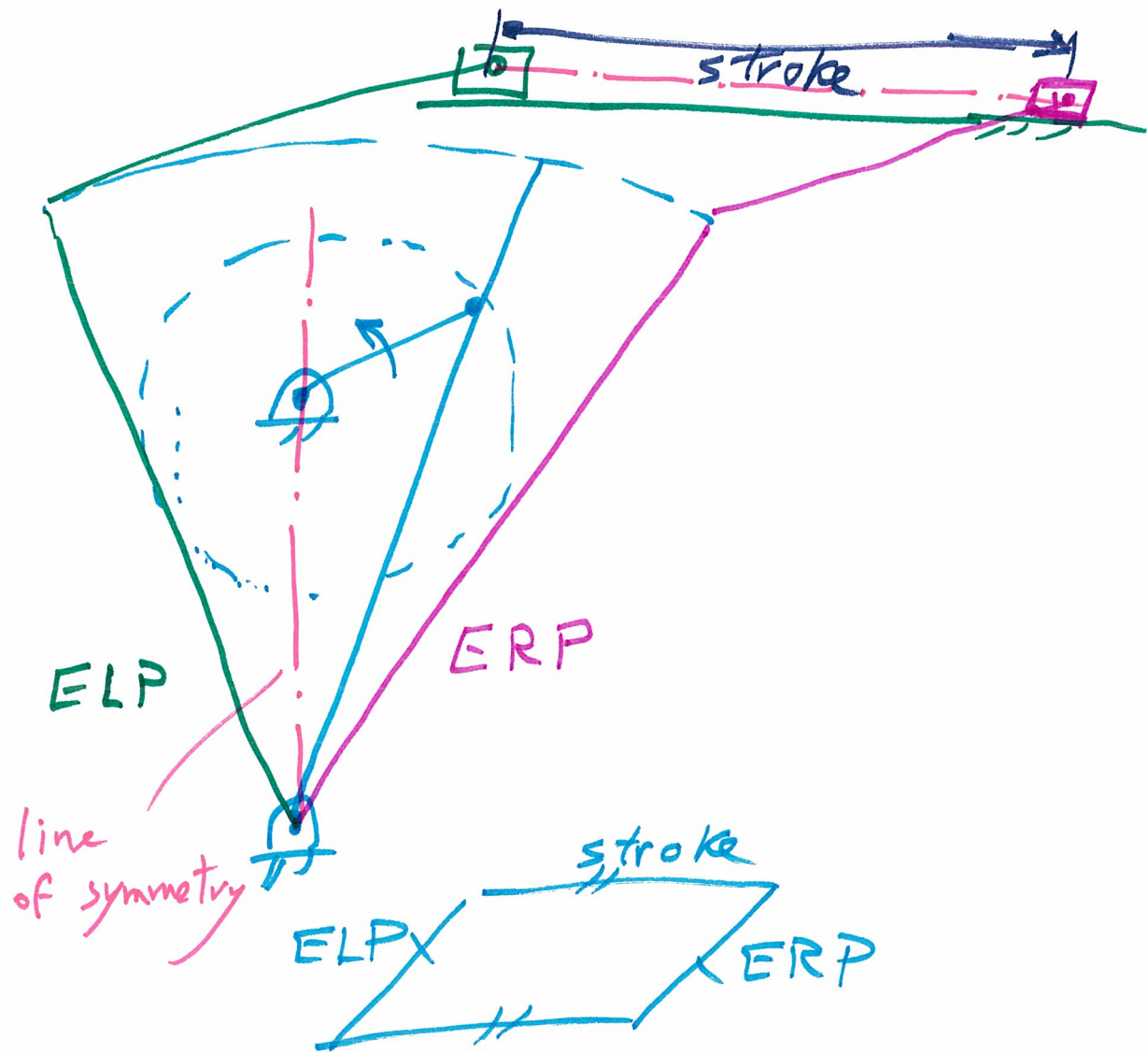
RRPRRRP

$$M = 3(6-1) - 2(7) = 1$$

≡



$$M = 3(5-1) - 2(5) - 1 = 12 - 10 - 1 = 1$$



Objective use this mechanism as a QRM

Design to produce specific time ratio (TR) and a stroke

start with the time ratio

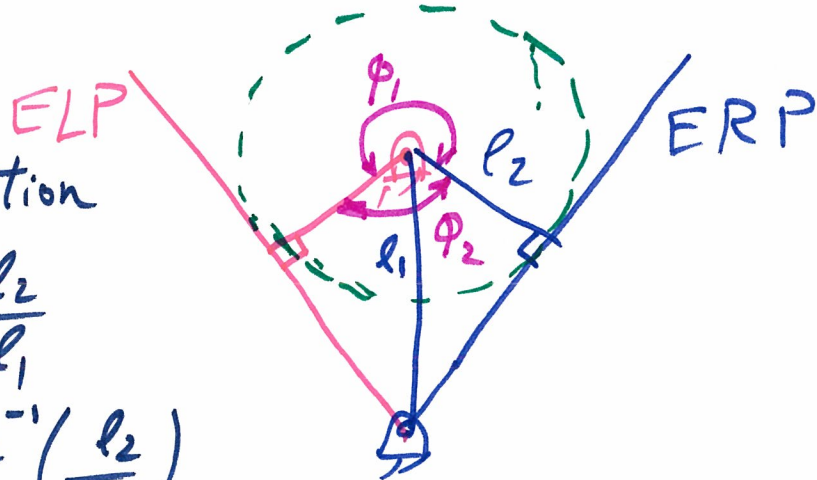
Identify ERP & ELP

by observation

$$\cos\left(\frac{\phi_2}{2}\right) = \frac{l_2}{l_1}$$

$$\phi_2 = 2 \cos^{-1}\left(\frac{l_2}{l_1}\right)$$

$$TR = \frac{\phi_1}{\phi_2} = \frac{360 - \phi_2}{\phi_2}$$



shaper
Example Design a QRM with time ratio = 2

$$2 = \frac{360 - \phi_2}{\phi_2} \rightarrow \phi_2 = 120^\circ$$

$$\cos\left(\frac{120}{2}\right) = \frac{l_2}{l_1} \rightarrow 0.5 = \frac{l_2}{l_1} \rightarrow 1 \text{ eqn. unk.}$$

Let $l_2 = 1''$
 $l_1 = 2''$

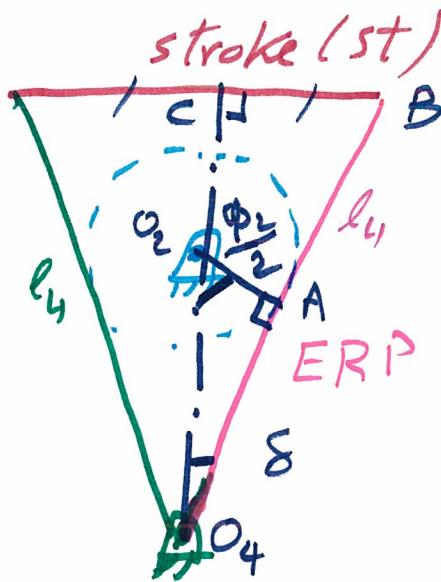
Stroke Calculations

From $\Delta O_2 O_4 A$

$$\delta = 90 - \frac{\phi_2}{2}$$

From $\Delta O_4 C B$

$$\frac{st}{2} = l_4 \sin \delta$$



$$st = 2 l_4 \sin\left(90 - \frac{\phi_2}{2}\right)$$

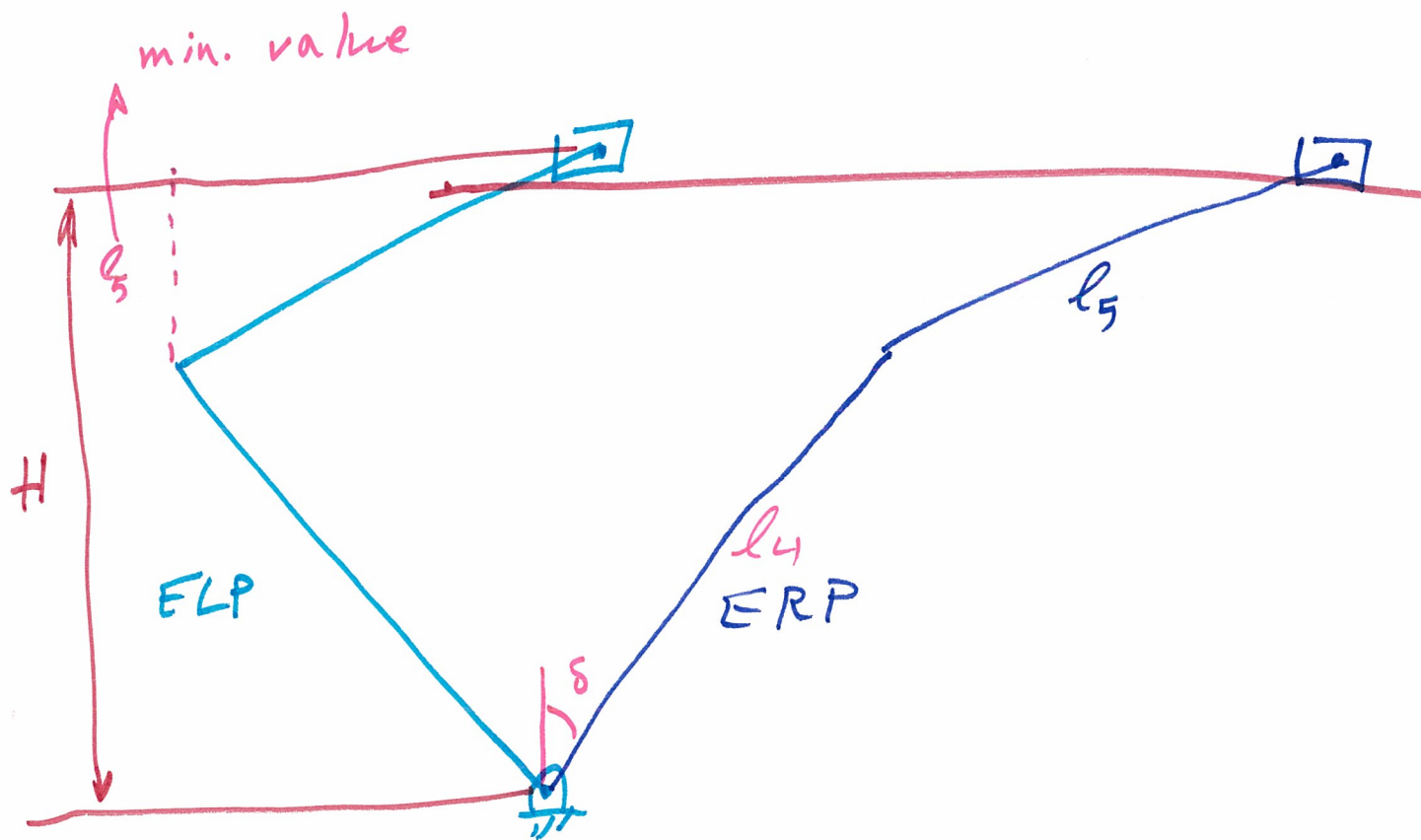
Continue the previous example:
~~Design~~ Design for a stroke = 4"

$$st = 2 l_4 \sin\left(90 - \frac{\phi_2}{2}\right)$$

$$4 = 2 l_4 \sin\left(90 - \frac{120}{2}\right)$$

$$4 = 2 l_4 \sin 30$$

$$l_4 = \frac{2}{\sin 30} = 4"$$



Assembly condition

$$l_4 \cos \delta + l_5 > H$$

$$l_4 \cos(90 - \frac{\phi_2}{2}) + l_5 > H$$

Finish the example

$$\text{Let } H = 6''$$

$$l_4 \cos(90 - \frac{\phi_2}{2}) + l_5 > H$$

$$4 \cos(90 - \frac{120}{2}) + l_5 > 6$$

$$l_5 > 2.8$$

$$\text{Let } l_5 = 4''$$

QRM Design Procedures

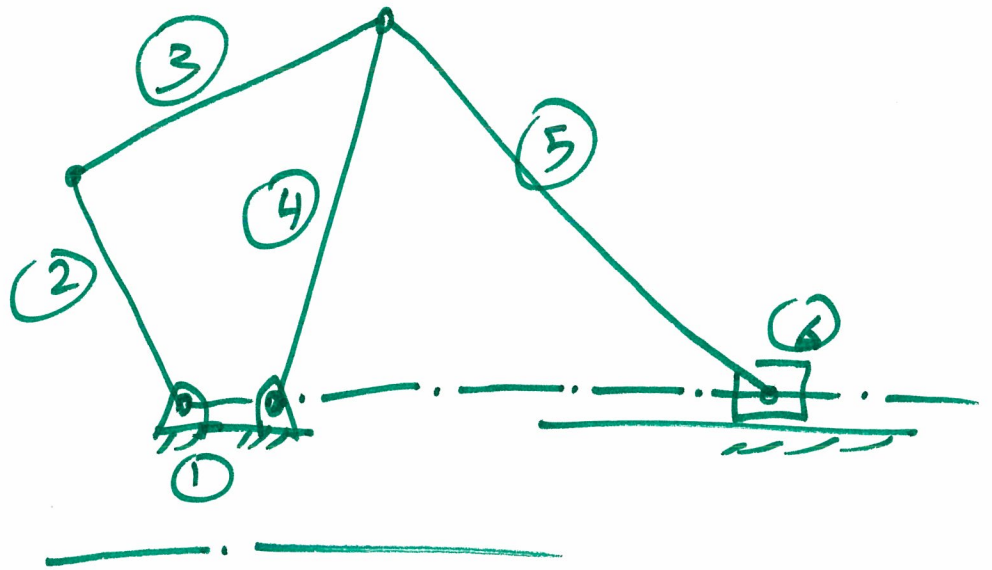
- ① Start by identifying extreme right & left positions
- ② Use these positions to define the angles of the crank (input link)
- ③ Find expressions relating ϕ_1 & ϕ_2 to ERP & ELP configurations
- ④ Consider the output motion which can be oscillation or translation and relate the output to the extreme positions

HW

Design a crank-crank crank-slider QRM with the following specifications.

Time ratio = 2

Stroke = 4"



HW

Simulate the above machine
using Working Model 2D